## ABQAYS EARNFNG

## STRUCTURAL ANALYSIS

## EI GHTH EDITION IN SI UNITS

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## CHAPTER 11:

## DISPLACEMENT METHOD OF ANALYSIS: SLOPE-DEFLECTION EQUATIONS



## Chapter Outline

11.1 Displacement Method of Analysis: General Procedures
11.2 Slope-Deflection Equations
11.3 Analysis of Beams
11.4 Analysis of Frames: No Sidesway
11.5 Analysis of Frames: Sidesway

## 11.1 <br> DISPLACEMENT METHOD OF ANALYSIS: GENERAL PROCEDURES



## Displacement Method of Analysis: General Procedures

- Displacement (disp) method requires satisfying equilibrium eqn for the structures
- The unknown disp are written in terms of the loads by using the load-disp relations
- These eqn are solved for the disp
- Once the disp are obtained, the unknown loads are determined from the compatibility eqn using the load-disp relations
- When a structure is loaded, specified points on it called nodes, will undergo unknown disp
- These disp are referred to as the degree of freedom
- The no. of these unknowns is referred to as the degree in which the structure is kinematically indeterminate
- We will consider some examples


## Displacement Method of Analysis: General Procedures

- Any load applied to the beam will cause node A to rotate
- Node B is completely restricted from moving
- Hence, the beam has only one unknown degree of freedom

- The beam has nodes at $A, B \& C$
- There are 4 degrees of freedom $\theta_{A}, \theta_{B}, \theta_{C}$ and $\Delta_{C}$


## 11.2 <br> SLOPE -DEFLECTION EQUATIONS



## Slope-Deflection Equations

- Slope deflection method requires less work both to write the necessary eqn for the solution of a problem \& to solve these eqn for the unknown disp \& associated internal loads
- General Case
- To develop the general form of the slope-deflection eqn, we will consider the typical span $A B$ of the continuous beam when subjected to arbitrary loading



## Slope-Deflection Equations

- Angular Displacement
- Consider node $A$ of the member to rotate $\theta_{A}$ while its end node $B$ is held fixed
- To determine the moment $M_{A B}$ needed to cause this disp, we will use the conjugate beam method



## Slope-Deflection Equations

- Angular Displacement

$$
\begin{aligned}
& \sum M_{A^{\prime}}=0 \\
& {\left[\frac{1}{2}\left(\frac{M_{A B}}{E I}\right) L\right] \frac{L}{3}-\left[\frac{1}{2}\left(\frac{M_{B A}}{E I}\right) L\right] \frac{2 L}{3}=0}
\end{aligned}
$$

$$
\sum M_{B^{\prime}}=0
$$

$$
\left[\frac{1}{2}\left(\frac{M_{B A}}{E I}\right) L\right] \frac{L}{3}-\left[\frac{1}{2}\left(\frac{M_{A B}}{E I}\right) L\right] \frac{2 L}{3}+\theta_{A} L=0
$$

## Slope-Deflection Equations

- Angular Displacement
- From which we obtain the following:

$$
M_{A B}=\frac{4 E I}{L} \theta_{A} \quad M_{B A}=\frac{2 E I}{L} \theta_{A}
$$

- Similarly, when end $B$ of the beam rotates to its final position while end $A$ is held fixed, we can relate the applied moment $\mathrm{M}_{\mathrm{BA}}$ to the angular disp $\theta_{\mathrm{B}}$ \& the reaction moment $\mathrm{M}_{\mathrm{AB}}$ at the wall

$$
M_{B A}=\frac{4 E I}{L} \theta_{B} \quad M_{A B}=\frac{2 E I}{L} \theta_{B}
$$



## Slope-Deflection Equations

- Relative Linear Displacement
- If the far node B of the member is displaced relative to A, so that the cord of the member rotates clockwise \& yet both ends do not rotate, then equal but opposite moment and shear reactions are developed in the member
- Moment M can be related to the disp using conjugate beam method

real beam


## Slope-Deflection Equations

- Relative Linear Displacement
- The conjugate beam is free at both ends since the real member is fixed supported
- The disp of the real beam at $B$, the moment at end $B^{\prime}$ of the conjugate beam must have a magnitude of $\Delta$ as indicated

$$
\begin{aligned}
& \sum M_{B^{\prime}}=0 \\
& {\left[\frac{1}{2} \frac{M}{E I}(L) \frac{2 L}{3}\right]-\left[\frac{1}{2} \frac{M}{E I}(L) \frac{L}{3}\right]-\Delta=0} \\
& M_{A B}=M_{B A}=M=\frac{-6 E I}{L^{2}} \Delta
\end{aligned}
$$


conjugate beam

## Slope-Deflection Equations

- Fixed end moments
- In general, linear \& angular disp of the nodes are caused by loadings acting on the span of the member
- To develop the slope-deflection eqn, we must transform these span loadings into equivalent moment acting at the nodes \& then use the loaddisp relationships just derived
- If the end moments due to each disp \& loading are added together, the resultant moments at the ends can be written as:

$$
\begin{aligned}
& M_{A B}=2 E\left(\frac{I}{L}\right)\left[2 \theta_{A}+\theta_{B}-3\left(\frac{\Delta}{L}\right)\right]+F E M_{A B}=0 \\
& M_{B A}=2 E\left(\frac{I}{L}\right)\left[2 \theta_{B}+\theta_{A}-3\left(\frac{\Delta}{L}\right)\right]+F E M_{B A}=0
\end{aligned}
$$



## Slope-Deflection Equations

- Fixed end moments
- The results can be expressed as a single eqn
$M_{N}=2 E k\left[2 \theta_{N}+\theta_{F}-3 \psi\right]+F E M_{N}$
$M_{N}=$ internal moment at the near end of the span
$E, k=$ modulus of elasticity $\&$ span stiffness
$\theta_{N}, \theta_{F}=$ near and far end slopes or angular disp of the span at the supports
$\psi=$ span rotation of its cord due to a linear disp
$F E M_{N}=$ fixed end moment at the near end support


## Slope-Deflection Equations

- Pin supported end span
- Sometimes an end span of a beam or frame is supported by a pin or roller at its far end
- The moment at the roller or pin is zero provided the angular disp at this support does not have to be determined

$$
\begin{aligned}
& M_{N}=2 E k\left[2 \theta_{N}+\theta_{F}-3 \psi\right]+F E M_{N} \\
& 0=2 E k\left[2 \theta_{N}+\theta_{F}-3 \psi\right]+0
\end{aligned}
$$



## Slope-Deflection Equations

- Pin supported end span
- Simplifying, we get:

$$
M_{N}=3 E k\left[\theta_{N}-\psi\right]+F E M_{N}
$$

- This is only applicable for end span with far end pinned or roller supported


## 11.3

ANALYSIS OF BEAMES


## Analysis of Beams

## Example 11.1

Draw the shear and moment diagrams for the beam where El is constant.


## Analysis of Beams

## Example 11.1 (Solution)

## Slope-deflection equations

2 spans must be considered in this problem Using the formulas for FEMs, we have:

$$
(F E M)_{B C}=-\frac{w L^{2}}{30}=-\frac{6\left(6^{2}\right)}{30}=-7.2 \mathrm{kN} \cdot \mathrm{~m}
$$

$$
(F E M)_{C B}=\frac{w L^{2}}{20}=\frac{6\left(6^{2}\right)}{20}=10.8 \mathrm{kN} \cdot \mathrm{~m}
$$

## Analysis of Beams

## Example 11.1 (Solution)

## Slope-deflection equations

- Note that $(F E M)_{B C}$ is -ve and
- $(\text { FEM })_{A B}=(F E M)_{B A}=0$ since there is no load on span $A B$
- Since A \& C are fixed support, $\theta_{A}=\theta_{C}=0$
- Since the supports do not settle nor are they displaced up or down, $\psi_{\mathrm{AB}}=\psi_{\mathrm{BC}}=0$

$$
\begin{align*}
& M_{N}=2 E\left(\frac{I}{L}\right)\left[2 \theta_{N}+\theta_{F}-3 \psi\right]+F E M_{N} \quad \mathbf{M}_{A B}\left(\frac{\mathbf{M}_{B A}}{4} \mathbf{M}_{B C}\right. \\
& M_{A B}=2 E\left(\frac{I}{8}\right)\left[2(0)+\theta_{B}-3(0)\right]+0=\frac{E I}{4} \theta_{B} \tag{1}
\end{align*}
$$

## Analysis of Beams

## Example 11.1 (Solution)

## Slope-deflection equations

Similarly,

$$
\begin{align*}
& M_{B A}=\frac{E I}{2} \theta_{B}  \tag{2}\\
& M_{B C}=\frac{2 E I}{3} \theta_{B}-7.2  \tag{3}\\
& M_{C B}=\frac{E I}{3} \theta_{B}+10.8 \tag{4}
\end{align*}
$$

## Analysis of Beams

## Example 11.1 (Solution)

## Equilibrium equations

The necessary fifth eqn comes from the condition of moment equilibrium at support B
Here $\mathbf{M}_{B A} \& \mathbf{M}_{B C}$ are assumed to act in the +ve direction to be consistent with the slope-deflection eqn

$$
\begin{equation*}
M_{B A}+M_{B C}=0 \tag{5}
\end{equation*}
$$



## Analysis of Beams

## Example 11.1 (Solution)

## Equilibrium equations

Sub eqn (2) and (3) into eqn (5) gives :
$\theta_{B}=\frac{6.17}{E I}$

Re - sub this value into eqn (1) to (4) gives :
$M_{A B}=1.54 \mathrm{kN} \cdot \mathrm{m} ; \quad M_{B A}=3.09 \mathrm{kN} \cdot \mathrm{m} ;$
$M_{B C}=-3.09 \mathrm{kN} \cdot \mathrm{m} ; M_{C B}=12.86 \mathrm{kN} \cdot \mathrm{m}$

## Analysis of Beams

## Example 11.1 (Solution)

## Equilibrium equations

- Using these results, the shears at the end spans are determined.
- The free-body diagram of the entire beam \& the shear \& moment diagrams are shown

11.4

ANALYSIS OF FRAMES: NO SIDESWAY


## Analysis of Frames: No Sidesway

- A frame will not sidesway to the left or right provided it is properly restrained
- No sidesway will occur in an unrestrained frame provided it is symmetric wrt both loading and geometry



## Analysis of Frames: No Sidesway

## Example 11.5

Determine the moments at each joint of the frame. El is constant.


## Analysis of Frames: No Sidesway

## Example 11.5 (Solution)

## Slope-deflection equations

3 spans must be considered in this case: $A B, B C \& C D$
$(F E M)_{B C}=-\frac{5 w L^{2}}{96}=-80 \mathrm{kN} \cdot \mathrm{m}$
$(F E M)_{C B}=\frac{5 w L^{2}}{96}=80 \mathrm{kN} \cdot \mathrm{m}$

Note that $\theta_{A}=\theta_{D}=0$ and $\psi_{A B}=\psi_{B C}=\psi_{C D}=0$

## Analysis of Frames: No Sidesway

## Example 11.5 (Solution)

## Slope-deflection equations

We have

$$
\begin{aligned}
& M_{N}=2 E\left(\frac{I}{L}\right)\left[2 \theta_{N}+\theta_{F}-3 \psi\right]+F E M_{N} \\
& M_{A B}=0.1667 E I \theta_{B} \\
& M_{B A}=0.333 E I \theta_{B} \\
& M_{B C}=0.5 E I \theta_{B}+0.25 E I \theta_{C}-80 \\
& M_{C B}=0.5 E I \theta_{C}+0.25 E I \theta_{B}+80 \\
& M_{C D}=0.333 E I \theta_{C} \\
& M_{D C}=0.1667 E I \theta_{C}
\end{aligned}
$$

## Analysis of Frames: No Sidesway

## Example 11.5 (Solution)

## Slope-deflection equations

The remaining 2 eqn come from moment equilibrium at joints $B \& C$, we have:

$$
\begin{aligned}
& M_{B A}+M_{B C}=0 \\
& M_{C B}+M_{C D}=0
\end{aligned}
$$

Solving for these 8 eqns, we get:


$$
\begin{aligned}
& 0.833 E I \theta_{B}+0.25 E I \theta_{C}=80 \\
& 0.833 E I \theta_{C}+0.25 E I \theta_{B}=-80
\end{aligned}
$$

## Analysis of Frames: No Sidesway

## Example 11.5 (Solution)

## Slope-deflection equations

Solving simultaneously yields:

$$
\theta_{B}=-\theta_{C}=\frac{137.1}{E I}
$$

$$
M_{A B}=22.9 \mathrm{kN} \cdot \mathrm{~m} ; \quad M_{B A}=45.7 \mathrm{kN} \cdot \mathrm{~m}
$$


$M_{B C}=-45.7 \mathrm{kN} \cdot \mathrm{m} ; ~ M_{C B}=45.7 \mathrm{kN} \cdot \mathrm{m}$
$M_{C D}=-45.7 \mathrm{kN} \cdot \mathrm{m} ; \quad M_{D C}=-22.9 \mathrm{kN} \cdot \mathrm{m}$

## 11.5

ANALYSIS OF FRAMES: SIDESWAY


## Analysis of Frames: Sidesway

- A frame will sidesway when it or the loading acting on it is nonsymmetric
- The loading $\mathbf{P}$ causes an unequal moments at joint $B \& C$
- $\mathbf{M}_{B C}$ tends to displace joint B to the right
- $\mathbf{M}_{C B}$ tends to displace joint C to the left
- Since $\mathbf{M}_{B C}>\mathbf{M}_{C B}$, the net result is a sidesway of both joint $B \& C$ to the right



## Analysis of Frames: Sidesway

- When applying the slope-deflection eqn to each column, we must consider the column rotation, $\psi$ as an unknown in the eqn
- As a result, an extra equilibrium eqn must be included in the solution
- The techniques for solving problems for frames with sidesway is best illustrated by e.g.


## Analysis of Frames: Sidesway

## Example 11.9

Explain how the moments in each joint of the two-story frame are determined.
El is constant.


## Analysis of Frames: Sidesway

## Example 11.9 (Solution)

## Slope-deflection equations

We have 12 equations that contain 18 unknowns
$M_{A B}=2 E\left(\frac{I}{5}\right)\left[2(0)+\theta_{B}-3 \psi_{1}\right]+0$
(1) $\quad M_{B E}=2 E\left(\frac{I}{7}\right)\left[2 \theta_{B}+\theta_{E}-3(0)\right]+0$
$M_{B A}=2 E\left(\frac{I}{5}\right)\left[2 \theta_{B}+0-3 \psi_{1}\right]+0$
(2) $\quad M_{E B}=2 E\left(\frac{I}{7}\right)\left[2 \theta_{E}+\theta_{B}-3(0)\right]+0$
$M_{B C}=2 E\left(\frac{I}{5}\right)\left[2 \theta_{B}+\theta_{C}-3 \psi_{2}\right]+0$
(3) $\quad M_{E D}=2 E\left(\frac{I}{5}\right)\left[2 \theta_{E}+\theta_{D}-3 \psi_{2}\right]+0$
$M_{C B}=2 E\left(\frac{I}{5}\right)\left[2 \theta_{C}+\theta_{B}-3 \psi_{2}\right]+0$
(4) $\quad M_{D E}=2 E\left(\frac{I}{5}\right)\left[2 \theta_{D}+\theta_{E}-3 \psi_{2}\right]+0$
$M_{C D}=2 E\left(\frac{I}{7}\right)\left[2 \theta_{C}+\theta_{D}-3(0)\right]+0$
(5) $\quad M_{F E}=2 E\left(\frac{I}{5}\right)\left[2(0)+\theta_{E}-3 \psi_{1}\right]+0$
$M_{D C}=2 E\left(\frac{I}{7}\right)\left[2 \theta_{D}+\theta_{C}-3(0)\right]+0$
(6) $\quad M_{E F}=2 E\left(\frac{I}{5}\right)\left[2 \theta_{E}+0-3 \psi_{1}\right]+0$

## Analysis of Frames: Sidesway

## Example 11.9 (Solution)

## Equilibrium equations

No FEMs have to be calculated since the applied loading acts at the joints
Members AB \& FE undergo rotations of $\psi_{1}=\Delta_{1} / 5$
Members $A B \& F E$ undergo rotations of $\psi_{2}=\Delta_{2} / 5$
Moment equilibrium of joints $B, C, D$ and $E$, requires

$$
\begin{align*}
& M_{B A}+M_{B E}+M_{B C}=0  \tag{13}\\
& M_{C B}+M_{C D}=0  \tag{14}\\
& M_{D C}+M_{D E}=0  \tag{15}\\
& M_{E F}+M_{E B}+M_{E D}=0 \tag{16}
\end{align*}
$$



## Analysis of Frames: Sidesway

## Example 11.9 (Solution)

## Equilibrium equations

Similarly, shear at the base of columns must balance the applied horizontal loads

$$
\begin{align*}
\sum F_{\mathrm{x}}=0 \Rightarrow & 40-\mathrm{V}_{\mathrm{BC}}-\mathrm{V}_{\mathrm{ED}}=0 \\
& 40+\frac{M_{B C}+M_{C B}}{5}+\frac{M_{E D}+M_{D E}}{5}=0  \tag{17}\\
\sum F_{\mathrm{x}}=0 \Rightarrow & 40+80-\mathrm{V}_{\mathrm{AB}}-\mathrm{V}_{\mathrm{FE}}=0 \\
& 120+\frac{M_{A B}+M_{B A}}{5}+\frac{M_{E F}+M_{F E}}{5}=0 \tag{18}
\end{align*}
$$

## Analysis of Frames: Sidesway

## Example 11.9 (Solution)

## Equilibrium equations

- Sub eqns (1) to (12) into eqns (13) to (18)
- These eqns can be solved simultaneously
- The results are resub into eqns (1) to (12) to obtain the moments at the joints


## Thank you

